

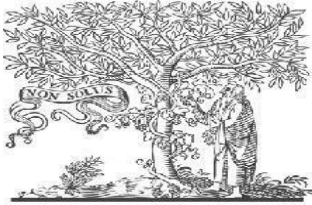


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Title: **DYNAMICS OF ELASTIC MEDIUM ON ACTION OF NON- STATIONARY CYLINDRICAL SOURCES**

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DYNAMICS OF ELASTIC MEDIUM ON ACTION OF NON- STATIONARY CYLINDRICAL SOURCES.

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Abstract: The article describes the phenomenon of "frame representations" and "blends", their functioning in the language, and provides an explanation of the degree and frequency of use of these units in modern Russian and modern cognition. The semantic analysis of these concepts is carried out.

Keywords: frame, frame representations, blends, frame technologies, script, attribute, slot, slot name, slot value, semantic networks, sample frames, instance frames, prototype, communication, subframes, concept.

Introduction

Consider the propagation of waves in an infinitely elastic medium (Fig. 1). The main goal is to obtain an accurate analytical expression of displacement and stress. Equation of motion of the environment of a cylindrical source

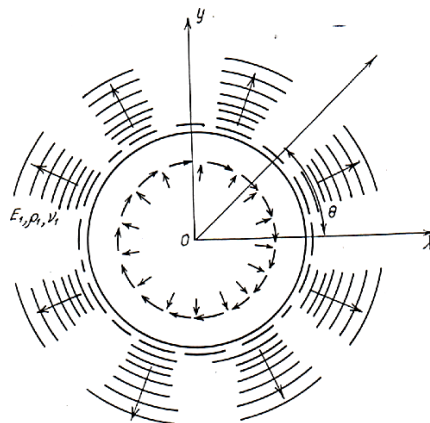


Fig. 1 Design scheme

$$\mu \nabla^2 \vec{u} + (\lambda + \mu) \text{grad} \text{div} \vec{u} = \rho \frac{\partial^2 \vec{u}}{\partial t^2} \quad (1)$$

in coordinates (r, θ) takes the form

$$\Delta \left(\mu \cdot \Delta^2 \cdot \psi - \rho \frac{\partial^2 \psi}{\partial t^2} \right) l_z + \Delta \left(\frac{E}{1-\nu^2} \Delta^2 \phi - \rho \frac{\partial^2 \phi}{\partial t^2} \right) = 0 \quad (2)$$

where

$$\vec{u} = \text{grad}_z \phi + \text{rot} \vec{\psi}$$

or

$$\vec{u} = V \vec{t} + V \vec{l}_0 \left(\frac{\partial \phi}{\partial r} + \frac{1}{r} \frac{\partial \psi}{\partial \theta} \right) \vec{l}_r + \left(\frac{1}{r} \frac{\partial \phi}{\partial \theta} - \frac{\partial \psi}{\partial r} \right) \vec{l}_\theta \quad (3)$$

Φ and ϕ are displacement potentials; $(\zeta_r) \rightarrow$ and $(\zeta_\theta) \rightarrow$ - unit vectors ($|(l_r) \rightarrow| = |(l_\theta) \rightarrow| = 1$). Displacement potentials (3) satisfy the wave equations, i.e.

$$\Delta^2 \phi = \frac{1}{c_1^2} \frac{\partial^2 \phi}{\partial t^2}, \Delta^2 \psi = \frac{1}{c_2^2} \frac{\partial^2 \psi}{\partial t^2} \quad (4)$$

where

$$C_1^2 = (\lambda + 2 \cdot \mu) / \rho, \quad C_2^2 = \mu / \rho$$

Initial condition $(\vec{u}|_{t=0}=0, \frac{d\vec{u}}{dt}|_{t=0}=0)$, (5) in the displacement potentials takes the form

$$\Phi(r, \theta, t)|_{t=0} = \psi(r, \theta, t)|_{t=0} = 0 \quad r > \alpha, -\pi < \theta \leq \pi \quad (6)$$

$$\frac{\partial \phi(r, \theta, t)}{\partial t} \Big|_{t=0} = \frac{\partial \psi(r, \theta, t)}{\partial t} \Big|_{t=0} = 0.$$

$r > \alpha, -\pi < \theta \leq \pi$

in a limitless environment $t=0$ transient load acts in the form

$$\left. \begin{aligned} \sigma_{rr}(\alpha, \theta, t) &= P_r(\theta, t), \\ \sigma_{\theta\theta}(\alpha, \theta, t) &= P_\theta(\theta, t) \end{aligned} \right\} \quad (7)$$

where $P_r(\theta, t)$ and $P_\theta(\theta, t)$ specified external load. The problem is solved in dimensionless coordinates.

$$(\sigma_r, \sigma_\theta, \sigma_{r\theta}) = (\sigma_r, \sigma_\theta, \sigma_{r\theta}) / \mu, \quad (\dot{u}, v) = (u, v) / a,$$

$$(\phi, \psi) = (\phi, \psi) / \alpha^2, \quad r = r / \alpha, \quad (t, T) = (t, T) c_2 / a,$$

$$C = c_1 / c_2 = (2 / (1 - \nu))^{1/2}; \quad c_2 = 1, \quad \theta = 1.$$

If the longitudinal and transverse potentials are known, it is possible to outstrip the stresses and displacements in a cylindrical (r, θ, z) coordinate system.

$$\left. \begin{aligned} \sigma_{rr} &= c^2 \cdot \Delta^2 \cdot \phi - r \cdot \left[\frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} + \frac{1}{r^2} \frac{\partial \psi}{\partial \theta} + \frac{1}{r^2} \frac{\partial^2 \psi}{\partial r \partial \theta} \right] \\ \sigma_{\theta\theta} &= (1 + \nu) \cdot C_1^2 \cdot \Delta^2 \cdot \phi - \sigma_{rr} \\ \sigma_{r\theta} &= -\Delta^2 \cdot \psi + 2 \cdot \left[\frac{1}{r} \frac{\partial \psi}{\partial r} + \frac{1}{r^2} \frac{\partial \psi^2}{\partial \theta^2} + \frac{1}{r^2} \frac{\partial \phi}{\partial \theta} + \frac{1}{r} \frac{\partial \phi^2}{\partial r \partial \theta} \right] \end{aligned} \right\} \quad (8)$$

External load $P_r(\theta, t)$ and $P_\theta(\theta, t)$ can be represented as:

$$P_\theta(\theta, t) = \sum_{n=0}^{\infty} \left\{ P_c^{(n)}(t) \cos n \cdot \theta + P_s^{(n)}(t) \sin n \cdot \theta \right\} \quad (9)$$

$$P_r(\theta, t) = \sum_{n=0}^{\infty} \left\{ S_c^{(n)}(t) \cos n \cdot \theta + S_s^{(n)}(t) \sin n \cdot \theta \right\}$$

$$\begin{aligned}
 P_c^{(n)}(t) &= \{q_c^{(n)} \cos n \cdot \Omega \cdot t - q_s^{(n)} \sin \cdot n \cdot \Omega \cdot t\} \cdot H(t) \\
 P_s^{(n)}(t) &= \{q_c^{(n)} \sin n \cdot \Omega \cdot t - q_s^{(n)} \cos \cdot n \cdot \Omega \cdot t\} \cdot H(t) \\
 S_c^{(n)}(t) &= \{r_c^{(n)} \cos n \cdot \Omega \cdot t - r_s^{(n)} \sin \cdot n \cdot \Omega \cdot t\} \cdot H(t) \\
 S_s^{(n)}(t) &= \{r_c^{(n)} \sin n \cdot \Omega \cdot t + r_s^{(n)} \cos \cdot n \cdot \Omega \cdot t\} \cdot H(t)
 \end{aligned}$$

$q_c^{(n)}, q_s^{(n)}, r_c^{(n)}$ и $r_s^{(n)}$ - amplitudes of external loads, $H(t)$ - Heaviside unit functions.

Wave equation (4) taking into account (6) is solved by the method of integral Laplace transform

$$\begin{aligned}
 f(p) &= \alpha \{f(t)\} = \int_0^\infty e^{-pt} \cdot f(t) dt, \quad (10) \\
 \Delta^2 \phi &= \frac{p}{c_1^2} \psi, \quad \Delta^2 \phi = \frac{p}{c_2^2} \psi
 \end{aligned}$$

The solution to equation (10) is expressed in terms of the modified Bessel functions

$$\begin{aligned}
 \phi(r, \theta, p) &= \sum_{n=0}^\infty K_n \left[\frac{p \cdot r}{c_1} \right] [A_n(p) \cos n\theta + B_n(p) \sin n\theta] \\
 \phi(r, \theta, p) &= \sum_{n=0}^\infty K_n(p, r) [C_n(p) \cos n\theta + D_n(p) \sin n\theta],
 \end{aligned} \quad (11)$$

Where $K_n(x)$ – modified Bessel functions,

A_n, B_n, C_n and D_n – arbitrary constants are determined from the contact conditions. After that, we determine the corresponding voltage, which takes the following form.

$$\begin{aligned}
 \sigma_{rr}(r, \theta, p) &= \sum_{n=0}^\infty \{ [A_n(p) \cdot A \cdot L_n(p, c, r) + D_n(p) + B \cdot L_n(p, c_2, r)] \cos n\theta + \\
 &\quad + [B_n(p) \cdot A \cdot L_n(p, c_1, r) - C_n(p) \cdot B \cdot L_n(p, c_2, r)] \sin n\theta \}, \\
 r_{r\theta}(r, \theta, p) &= \sum_{n=0}^\infty \{ [B_n(p) \cdot B \cdot L_n(p, c_1, r) - C_n(p) \cdot A \cdot L_n(p, c_2, r)] \cos n\theta - \\
 &\quad - A_n(p) \cdot A \cdot L_n(p, c_1, r) \cos n\theta - [A_n(p) \cdot B \cdot L_n(p, c_2, r) + \\
 &\quad + D_n(p) \cdot A \cdot L_n(p, c_2, r)] \sin n\theta \}, \\
 \sigma_{\theta\theta}(r, \theta, p) &= (1 + \nu) \sum_{n=0}^\infty K_n \left(\frac{p \cdot r}{c_1} \right) [A_n \cdot \frac{p \cdot r}{c_1} \cos n\theta + B_n \cdot \frac{p \cdot r}{c_1} \sin n\theta] - \sigma_{rr}(r, \theta, p),
 \end{aligned} \quad (12)$$

where

$$AL_n(p, c_p, r) = [p^2 + \frac{2n(n-1)}{r^2}] K_n \left[\frac{p \cdot r}{c_p} \right] + \frac{2p}{c_p} K_{n+1} \left[\frac{p \cdot r}{c_p} \right]$$

$$BL_n(p, c_p, r) = \frac{2n}{c_p} \left[\frac{p \cdot r}{c_p} \right] K_n \left(\frac{p \cdot r}{c_p} \right) - \frac{p}{c_p} K_{n+1} \left(\frac{p \cdot r}{c_p} \right),$$

$$A_n(p) = \frac{[AL_n(p, c_2, a) p^n + BL_n(p, c_2, a) S_c^n(p)]}{[AL_n(p, c_2, a) p^n + BL_n(p, c_2, a) S_s^n(p)]},$$

$$B_n(p) = \frac{[AL_n(p, c_2, a) p^n - BL_n(p, c_2, a) S_c^n(p)]}{[AL_n(p, c_2, a) p^n + BL_n(p, c_2, a) S_s^n(p)]},$$

$$C_n(p) = \frac{[BL_n(p, c_1, a) p^n - AL_n(p, c_1, a) S_c^n(p)]}{[BL_n(p, c_1, a) p^n + AL_n(p, c_1, a) S_s^n(p)]},$$

$$D_n(p) = \frac{[BL_n(p, c_1, a) p^n + AL_n(p, c_1, a) S_s^n(p)]}{[BL_n(p, c_1, a) p^n + AL_n(p, c_1, a) S_s^n(p)]},$$

$$\Delta_n(p) = AL(p, c_1, a) AL(p, c_2, a) - BL(p, c_1, a) BL(p, c_1, a)$$

Passing from mapping (12) to the original, using the inversion theorem after mathematical transformations, we obtain the expressions for the stresses

$$\sigma_{rr}(r,\theta,t) = \sum_{k=1}^{\infty} r_{1k}(c_1, c_2, a, p_k) \cdot r_k(t)$$

$$\sigma_{r\theta}(r,\theta,t) = \sum_{k=1}^{\infty} r_{2k}(c_1, c_2, a, p_k) \cdot r_k(t)$$

$$\sigma_{\theta\theta}(r,\theta,t) = \sum_{k=1}^{\infty} r_{3k}(c_1, c_2, a, p_k) \cdot r_k(t)$$

$$r_k(t) = \int_0^{\infty} p(r)A(t-r)dr$$

$$r_{3k}(c_1, c_2, a, p_k) = -\frac{1}{p_k} \frac{\Delta_{nk} S_{mk} + \Delta_{nk} S_{mk}}{\left(\frac{\partial \Delta}{\partial p}\right)_{p=p_k}}$$

Where are the coefficients S_{mk} , Δ_{nk} and Δ_{nk} expressed in terms of special Bessel and Hankel functions of the first and second kind of the n th order.

In a particular case, consider the propagation of radial waves about cylindrical holes, then the equation of motion (4) takes the following form:

$$\frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} - \frac{u}{r^2} = \frac{1}{a^2} \frac{\partial^2 u}{\partial t^2}; \quad (13)$$

where $a^2 = (\lambda + 2\mu) / \rho$

Cylindrical surfaces $r = a$ at $t \geq 0$ are subjected to unsteady loads in the form

$$(\sigma_{rr})_{r=a} = -p\delta(t-T), \quad (14)$$

Where $\delta(t-T)$ – delta - Dirac functions. To solve equation (13), the integral Laplace transform (9) is also used and takes the following form:

$$\left. \begin{aligned} \frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} - \left(\frac{p^2}{a^2} + \frac{1}{r^2}\right) u &= 0 \\ [(\lambda + 2 \cdot \mu) \frac{\partial u}{\partial r} + \lambda \frac{u}{r}]_{r=a} &= -p \cdot e^{-pt} \end{aligned} \right\} (15)$$

Solution (15) is also expressed in terms of the cylindrical Bessel functions of the first and second kind

$$\begin{aligned} U(r, p) &= \frac{p \cdot e^{-pt} K_1\left(\frac{pr}{a}\right)}{p \cdot e^{-pt} K_0\left(\frac{pa}{a}\right) + \frac{2\mu}{a} K_1\left(\frac{pa}{a}\right)} \\ \sigma(r, p) &= \frac{p \cdot e^{-pt} (\lambda + 2\mu) \frac{p}{a} K_0\left(\frac{pr}{a}\right) + \frac{2\mu}{a} K_1\left(\frac{pr}{a}\right)}{p \cdot e^{-pt} (\lambda + 2\mu) \frac{pa}{a} K_0\left(\frac{pa}{a}\right) + \frac{2\mu}{a} K_1\left(\frac{pa}{a}\right)} \end{aligned} \quad (16)$$

In order to find the original, we use the following modified Bessel expression

$$K_n(z) = \frac{F\left(\frac{1}{2}\right)}{\Gamma\left(n + \frac{1}{2}\right)} \cdot \left(\frac{z^2}{2}\right) \int_1^{\infty} e^{-zt} (t^2 - 1)^{n-\frac{1}{2}} dt \quad (17)$$

After some transformations (16), using (17), the following expressions for displacements and stresses can be obtained:

$$U(r,t) = \frac{\alpha \cdot P \cdot e^{-PT}}{\alpha(\lambda+r\mu)} K_1(pr) \frac{e^p}{[K_0(p) + \frac{y}{p} K_1(p)] e^p}$$

$$\sigma_{rr}(r,p) = -p \cdot e^{-PT} [K_0(p,r) + \frac{y}{pr} K_1(pr)] \frac{e^p}{[K_0(p) + \frac{y}{p} K_1(p)] e^p}$$

The results of the work were used to determine the original.

$$\int_0^e u(r, \eta, T) \cdot q(t - \eta) d\eta = \begin{cases} \text{Для } 0 < t - T < r - 1 \\ \frac{a \cdot P}{a \cdot (\lambda + 2\mu)} \cdot \frac{1}{r} \sqrt{(t - T + 1)^2 - r^2} \end{cases}$$

для $t - T > r - 1$

$$\int_0^e \sigma_{rr}(r, \eta, T) \cdot q(t - \eta) d\eta = \begin{cases} 0, 0 < t - T < r - 1 \\ -P \frac{r^2 + y((t - T + 1)^2 - r^2)}{r^2 \sqrt{(t - T)^2 - r^2}} \quad t - T > r - 1 \end{cases}$$

где

$$g(t) = L^{-1} [e^p K_0(p) + \frac{e^p}{p} y K_1(p)] = (1 + y((t+1)^2 - 1)) / \sqrt{(t+1)^2 - 1}$$

let the load q be applied to the cylindrical hole at $t \geq 0$

$$(\sigma_{rr})_{r=a} = -q(t),$$

then the expression for displacement and stress takes the form:

$$\int_0^e u(r, \eta, T) \cdot q(t - \eta) d\eta = \begin{cases} \text{Для } 0 < t - T < r - 1 \\ \frac{1}{a \cdot (\lambda + 2\mu)} \cdot \frac{2 \cdot q}{2r} \cdot \frac{t+1}{2r} \sqrt{(t+1)^2 - r^2} - r/2 \cdot \\ \left[\log \left(\frac{t+1 + \sqrt{(t+1)^2 - r^2}}{r} \right) \right] \end{cases}$$

for $t > r - 1$

$$\int_0^e \sigma_{rr}(r, \eta, T) \cdot q(t - \eta) d\eta$$

0, for $0 < t - T < r - 1$

$$= \left\{ -q \left[\frac{t+1-y}{2r^2} \sqrt{(t+1)^2 - r^2} + \left(1 - \frac{y}{2}\right) \cdot \log \left(\frac{t+1 + \sqrt{(t+1)^2 - r^2}}{r} \right) \right] \right\} \text{ for } t > r - 1$$

Where $y = 2\mu/(\lambda + 2\mu)$

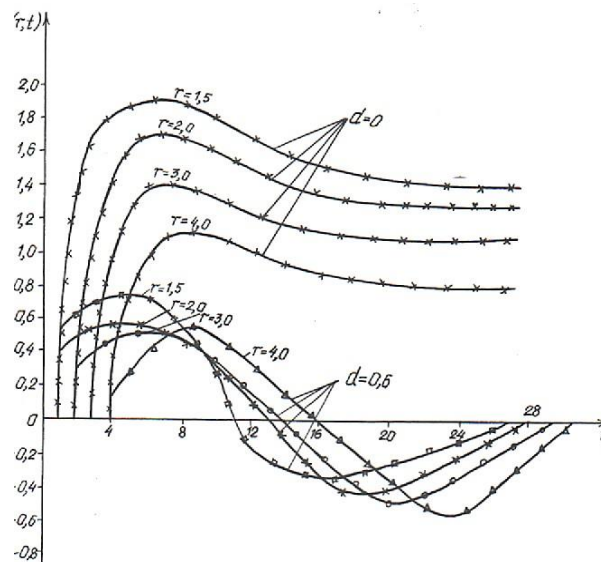
If a load is applied to the cylindrical hole in the form

$$\sigma_{rr}(r,t) = P_0 e^{-dT}$$

then, as in the first case, the displacement and stress can be found using the Duhamel integral.

$$\frac{a(\lambda+r\mu)}{a \cdot P_0} \int_{r-1}^t q(t-z) u(r,\eta) d\eta = \int_0^{t-r+1} \frac{1}{r} \sqrt{(t-T+1)^2 - r^2} e^{-dT} dT \int_{r-1}^t q(t-\phi) y(r,z) dz = \int_0^{t-r+1} \frac{r^2 + y\{(t-T+1)^2 - r^2\}}{r^2 \sqrt{(t-T+1)^2 - r^2}} e^{-dT} dT$$

The definitions of displacements and stresses in calculations are presented in Fig. 2 at $r = 1,5; 2,0; 3,0; 4,0$; $y=0,8$.



Rice. 2 Change of displacements of a cylindrical strip depending on time.

The calculation results show that displacements and stresses reach their maximum values in the initial period of time. Now let us determine the stress-strain state of a cylindrical hole in the two-dimensional case.

$$\left. \begin{aligned} y_{rr}(a, u, t) &= PH(t) \\ y_{ru}(a, u, t) &= 0 \end{aligned} \right\}$$

Here

$$P = \left\{ \frac{P_0}{r_0} (1 + \cos \frac{\pi\theta}{\theta_0}) (\theta + \Omega t) \right\}, |\theta| \leq Q_0$$

$$Q_0 < |\theta| < \pi$$

The expression $q_c(n)$ takes the following form

$$q_c^{(n)} = \begin{cases} \frac{P_0}{n} \sin \frac{\theta_0(\frac{\pi}{\pi^2 - n^2 \cdot \theta^2})}{2\pi}, & n \geq 1 \\ \frac{P_0 \cdot \theta_0}{2\pi}, & n = 0, n = \pi/\theta_0 \end{cases} \quad n = \frac{\pi}{\theta_0}$$

$$q_3^{(n)} = 0, K_c^{(n)} = r_0^{(n)} = 0$$

The calculation results are shown in Fig. 3,4,5.

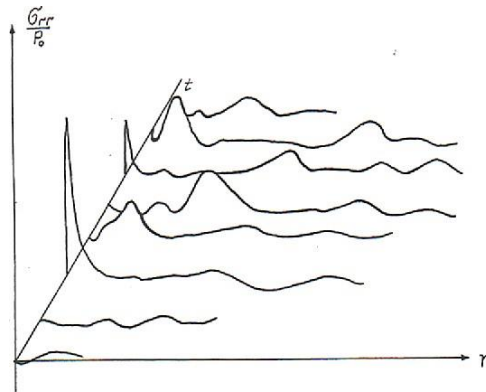


Fig. 3 Change in radial stress as a function of time at

$$\theta = 0$$

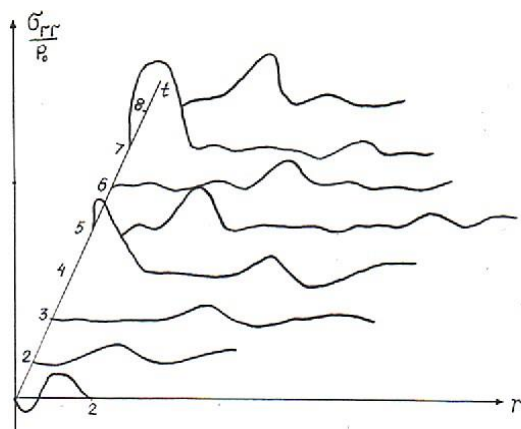


Fig. 4 Change in radial stress at $\theta = \frac{\pi}{3}$

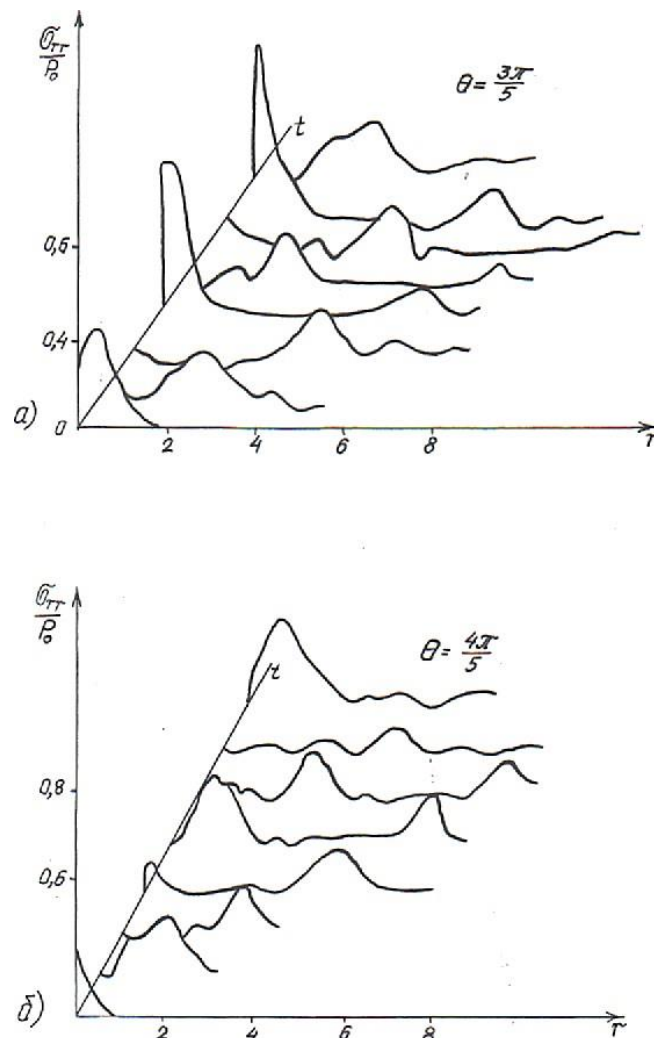


Fig. 5 Voltage change depending on time and coordinates.

($\theta=0, \pi/3; \pi/6$) depending on the time. It can be seen that all force factors take on a maximum value at the initial moment of time. Thus, we have obtained an accurate analytical expression of displacement and stress.

Obtaining numerical results in a particular case (for axially symmetric radial loads) is compared with the results of work [2], the results of calculations at the initial time differ by up to 20% with increasing time differs from 5% -15%.

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